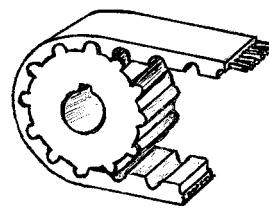


Polyurethane Timing Belts



Polyurethane Timing Belts are offered in three standard metric pitches, 2.5mm, 5mm, and 10mm, and also standard imperial pitch, $\frac{1}{5}$ " (XL). In addition to the standard single sided belts, double sided belts with moulded teeth on both sides of the belt can be supplied (T5 and T10 only) for multishaft, serpentine drives and some conveyor applications. The metric belts are offered in two designs 'T' and 'AT' series both using steel tension cords encased in the polyurethane jacket with integral drive teeth. The method of manufacture ensures close control of pitch length, which combined with the inelastic properties of the steel tension member create a belt drive with high positional accuracy resulting in these belts being popular for instrument drives, robotics, and servo mechanisms. The imperial pitch belts use Kelvar tension members for increased strength and flexibility making them suited to higher power applications. The metric series belts can be also supplied with Kelvar tension member if required. Polyurethane has excellent resistance to mineral oils, greases and many slight acidic solutions, it is basically non marking and resistant to crumbling making it suitable for food and cigarette processing machines, and for paper handling in office equipment. Polyurethane belts can be used on applications with environmental temperature range -30°C, to 80°C, with belt speed up to 80 m/sec. In addition to the standard belt listed on page 39, open ended belt can be supplied for most constructions and widths, and fitted with welded attachments for conveying applications and positional rack drives, refer to page 45 for further details. The low inertia of the belts and aluminium pulleys plus accurate pitching make both the endless and open ended belt ideal for the high acceleration rates encountered in robotics. The high flexibility of the polyurethane belt enables crossed drives to be achieved (shafts at right angles), where reasonable length centres exist and narrow width belts are used. Both high shaft speeds and power capabilities can be achieved with the 'T' and 'AT' series of belts. The T2.5, T5, and AT5 belts are all able to run at up to 40,000 r.p.m with AT5 belts transmitting 15kW. The T10 and AT10 belts, can operate to 15,000 r.p.m with power capacity to 70kW, and higher powers of up to 200kW can be achieved with 20mm pitch AT 20 belts available to special order.

AT Series Polyurethane Timing Belts

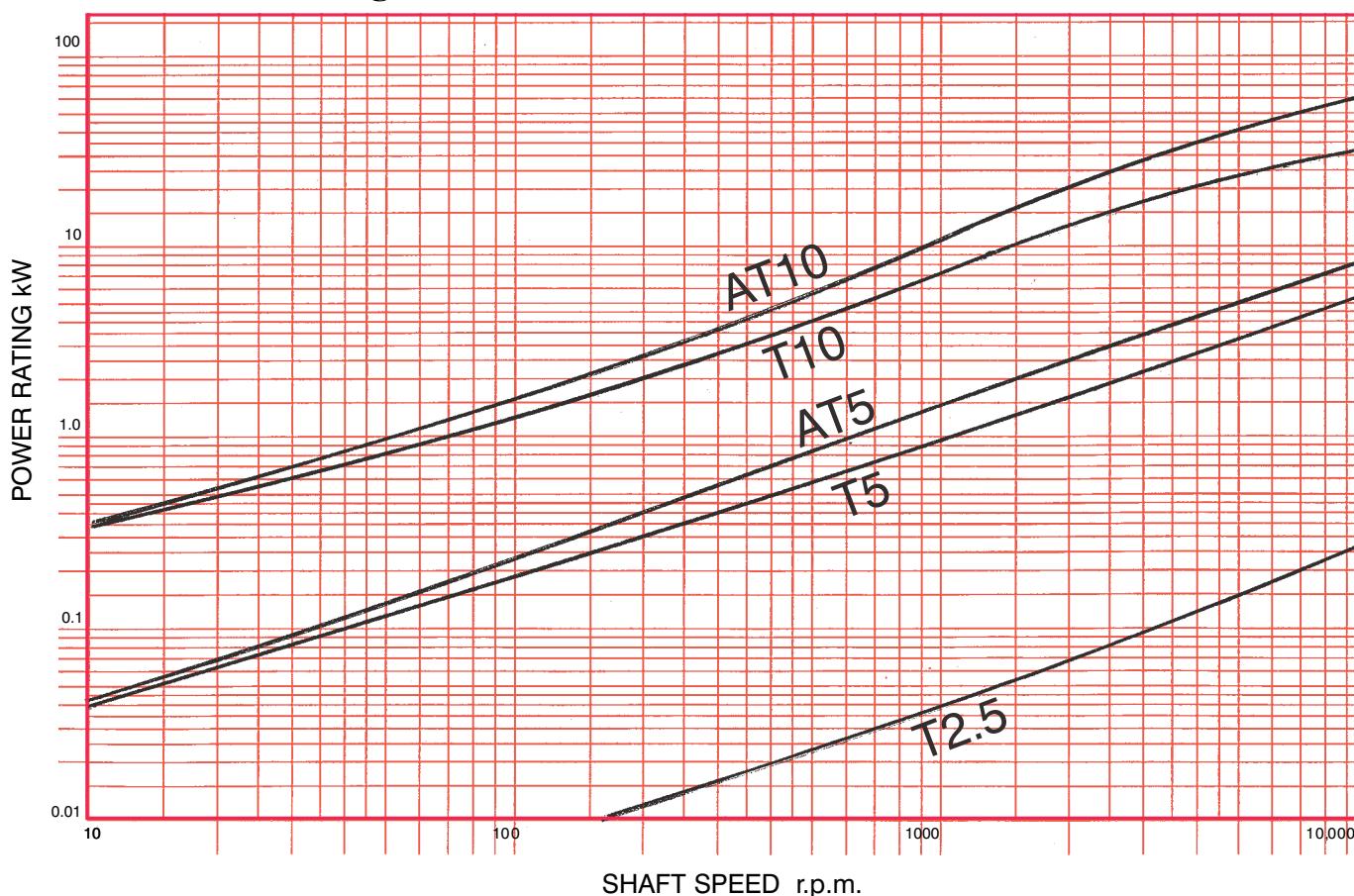


Polyurethane belts are suitable for drives in a wide variety of machinery including Office machinery, Machine Tools, Pumps, Textile Machinery, Printing Machinery, Paper manufacturers, down to precision Camera drives and servo mechanisms. The 'AT' series of Polyurethane belts have increased tooth width and higher strength tension members than the 'T' Series. The increased tooth size with resultant increased stiffness improves meshing with pulley teeth and enables transmission of higher powers. Increased strength tension members improve pitch accuracy and also increase power capacity. Both improvements result in an increase of power transmission capacity of approx 50%. Quieter operation, as a result of improved tooth meshing, and reduced polygonal effect plus ability to use narrower section belts, is combined with improved positional accuracy of power transmission, with linear accuracy better than $\pm 0.1\text{mm/Metre}$ belt length.

Design limits for standard Polyurethane Timing Belts.

| Belt Size Width x Pitch | Max. Allowable Belt Tension N. | Min. No. Teeth Drive Pulley | Min dia of Idler Pulley mm |
|----------------------------|-----------------------------------|--------------------------------|-------------------------------|
| 6T2.5 | 65 | 10 | 15 |
| 10T5 | 330 | 10 | 30 |
| 16T5 | 570 | 10 | 30 |
| 25T5 | 930 | 10 | 30 |
| 16T10 | 1100 | 12 | 60 |
| 25T10 | 1800 | 12 | 60 |
| 32T10 | 2300 | 12 | 60 |
| 50T10 | 3800 | 12 | 60 |
| 10AT5 | 490 | 15 | 60 |
| 16AT5 | 840 | 15 | 60 |
| 25AT5 | 1100 | 15 | 60 |
| 25AT10 | 3500 | 15 | 120 |
| 32AT10 | 4750 | 15 | 120 |
| 50AT10 | 7750 | 15 | 120 |

Standard Belt Power Ratings



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Polyurethane Belt Drives Selection Procedure



The most important factor considered when selecting a Polyurethane "T" or "AT" series belt is the tooth shear strength. The calculation of Power capacity is based on the specific shear strength of each tooth in mesh relative to the belt width in cms. The maximum tooth shear strength F_s must not be exceeded. Values for tooth shear strength are shown in the table at foot of the page. The "AT" series belts offer a higher tooth strength due to the larger tooth cross section. The high pitch accuracy of Polyurethane belts allows for up to 12 teeth to share the drive loads. Other factors which should be considered in belt selection are the number of teeth on the small pulley and diameters of tensioners/idlers pulleys. The drive design should also ensure that the maximum working load does not exceed maximum allowable tension of the members F_m .

In order to make a selection it is first necessary to compile together the following relevant design parameters

- a) Power to be transmitted P. kW
- b) Speed of fastest shaft N1 r.p.m.
- c) Drive ratio required, i - reduction or speed increase.
- d) Maximum pulley diameters which can be accommodated.
- e) Type of driver and driven equipment.
- f) Shaft diameters and centre distance A

Selection of Belt Pitch and Width.

- 1) The size of Belt selected must always ensure maximum tooth shear and tensile strengths are not exceeded, and that pulley sizes meet their criteria. Under start-up conditions normal running torques can be exceeded by 2-2.5 times with electric motors, and this must be allowed for in the calculation. Peak loads caused by oscillating and torsional loads can be up to 1.7 times mean torque loads, and design factor f_1* should consider this. Emergency braking systems may impose the maximum torque in the application. Speed increase drives impose heavier shock conditions, and a factor needs to be applied to cover these as below :-

$$\begin{array}{ll} i = 1 \text{ to } 1.5 & f_1 = 1.1 \\ i = 1.5 \text{ to } 2.5 & f_1 = 1.2 \\ i = 2.5 + & f_1 = 1.3 \end{array}$$

* Values for f_1 can be found on page 4 , Table 1.

The total design factor $f_d = f_1 + f_2$ or start-up overload factor f_s which ever is highest

$$\text{The design Power } P_d = P \times f_d \text{ or } f_s$$

Using graph on page opposite select suitable belt size to transmit the design power at the shaft speed N1 r.p.m.

2. Select number of teeth in pulleys, by consideration of the restraints of maximum pulley diameter and shaft diameters. The minimum pulley pitch diameter should be at least twice shaft diameter. The minimum number of teeth on the pulleys is also constrained by the belt design, reference belt characteristics table opposite page.
- The actual pulley diameters can be obtained by referring to pulley dimension tables on pages 40-43 or by using formula.

$$\text{Pitch dia pulley } dp = \frac{Z \cdot p}{\Pi} \text{ mm}$$

The number of teeth in small and large pulleys can be determined from the drive reduction ratio i.

$$i = \frac{Z_2}{Z_1}$$

- 3) Determine the number of teeth in mesh on small pulley Z_m from formula

$$Z_m = \frac{Z_1}{2\Pi} \left(\Pi - \sin^{-1} \left[\frac{Z_2 - Z_1}{\Pi \cdot A} \right] \right)$$

- 4) Determine belt tension from drive Power P_d or drive torque M_d

$$\begin{aligned} F_t &= \frac{1000 \cdot P_d}{Z_1 \cdot N_1 \cdot p} && \text{from power input} \\ \text{or } F_t &= \frac{2000 \cdot T_d}{dp} && \text{from torque input} \end{aligned}$$

- 5) Determine belt width by consideration of belt tooth shear strength

$$\text{belt width } b = \frac{F_t}{F_s \cdot Z_m} \text{ cm}$$

Select the next largest standard width for belt.

If result from below formula for belt width gives impractical result rework selection sequence with next sizes of belt to obtain revised width.

- 6) Final check belt tension maximum F_m is not exceeded i.e.

$$F_m \geq F_t$$

- 7) To determine belt length refer to paragraph 5 on page 3.

Terms and Definitions :-

| | | |
|----|--|-------|
| A | = centre distance pulley shafts | mm |
| b | = belt width | cm |
| d | = bore of pulley | mm |
| dp | = pitch diameter of pulley | mm |
| fd | = design factor | |
| fs | = starting overload factor | |
| Fm | = max. working tension in belt | N |
| Fs | = tooth shear resistance (see table below) | N/cm |
| FT | = total linear force on belt | N |
| i | = drive ratio | |
| L | = belt Length | mm |
| N1 | = shaft speed - high speed shaft | rp.m. |
| N2 | = shaft speed - low speed shaft | rp.m. |
| P | = motor power | k.w |
| Pd | = design power | k.w |
| P | = belt pitch | mm |
| Z1 | = no teeth on small pulley | |
| Z2 | = no teeth on large pulley | |
| Zm | = no teeth in mesh in small pulley | |

| Pulley Speed N r.p.m | Value for Teeth Shear Resistance F_s N/cm | | | | |
|-------------------------|---|-------|-------|------|------|
| | T2.5 | T5 | T10 | AT5 | AT10 |
| 0 | 9.03 | 24.00 | 50.50 | 35.3 | 73.5 |
| 20 | 8.72 | 23.38 | 49.00 | 34.9 | 72.4 |
| 40 | 8.48 | 22.86 | 47.70 | 34.5 | 71.4 |
| 60 | 8.28 | 22.41 | 46.60 | 34.1 | 70.5 |
| 80 | 8.10 | 22.01 | 45.70 | 33.8 | 69.6 |
| 100 | 7.95 | 21.65 | 44.80 | 33.5 | 68.7 |
| 200 | 7.39 | 20.28 | 41.40 | 32.0 | 65.0 |
| 300 | 7.01 | 19.30 | 39.10 | 30.9 | 62.1 |
| 400 | 6.71 | 18.55 | 37.20 | 29.8 | 59.5 |
| 500 | 6.48 | 17.93 | 35.70 | 29.0 | 57.4 |
| 600 | 6.28 | 17.41 | 34.40 | 28.2 | 55.5 |
| 700 | 6.11 | 16.96 | 33.30 | 27.5 | 53.7 |
| 800 | 5.97 | 16.56 | 32.40 | 26.8 | 52.2 |
| 900 | 5.83 | 16.20 | 31.50 | 26.3 | 50.8 |
| 1000 | 5.71 | 15.88 | 30.70 | 25.7 | 49.5 |
| 1100 | 5.61 | 15.58 | 30.00 | 25.2 | 48.3 |
| 1200 | 5.51 | 15.31 | 29.30 | 24.8 | 47.2 |
| 1300 | 5.41 | 15.06 | 28.70 | 24.3 | 46.2 |
| 1400 | 5.33 | 14.83 | 28.20 | 23.9 | 45.2 |
| 1500 | 5.25 | 14.61 | 27.60 | 23.5 | 44.3 |
| 1600 | 5.17 | 14.40 | 27.10 | 23.2 | 43.4 |
| 1700 | 5.10 | 14.21 | 26.70 | 22.8 | 42.6 |
| 1800 | 5.04 | 14.03 | 26.20 | 22.5 | 41.8 |
| 1900 | 4.97 | 13.85 | 25.80 | 22.2 | 41.0 |
| 2000 | 4.91 | 13.69 | 25.40 | 21.9 | 40.3 |
| 2200 | 4.80 | 13.38 | 24.60 | 21.3 | 39.0 |
| 2400 | 4.70 | 13.10 | 23.90 | 20.8 | 37.8 |
| 2600 | 4.60 | 12.84 | 23.30 | 20.3 | 36.6 |
| 2800 | 4.51 | 12.59 | 22.70 | 19.8 | 35.5 |
| 3000 | 4.43 | 12.37 | 22.20 | 19.4 | 34.5 |
| 3200 | 4.36 | 12.16 | 21.70 | 19.0 | 33.6 |
| 3400 | 4.28 | 11.96 | 21.20 | 18.6 | 32.7 |
| 3600 | 4.22 | 11.77 | 20.70 | 18.3 | 31.9 |
| 3800 | 4.15 | 11.59 | 20.30 | 17.9 | 31.1 |
| 4000 | 4.09 | 11.42 | 19.86 | 17.6 | 30.3 |
| 4500 | 3.95 | 11.03 | 18.91 | 16.9 | 28.5 |
| 5000 | 3.82 | 10.68 | 18.06 | 16.2 | 26.9 |
| 5500 | 3.70 | 10.36 | 17.28 | 15.6 | 25.5 |
| 6000 | 3.60 | 10.07 | 16.58 | 15.0 | 24.2 |
| 6500 | 3.51 | 9.81 | 15.93 | 14.5 | 23.0 |
| 7000 | 3.42 | 9.56 | 15.33 | 14.0 | 21.8 |
| 7500 | 3.33 | 9.33 | 14.76 | 13.5 | 20.8 |
| 8000 | 3.26 | 9.11 | 14.24 | 13.1 | 19.8 |
| 9000 | 3.11 | 8.72 | 13.28 | 12.3 | 17.9 |
| 10000 | 2.99 | 8.37 | 12.42 | 11.6 | 16.3 |
| 12000 | 2.77 | - | - | - | - |
| 15000 | 2.50 | - | - | - | - |
| 18000 | 2.28 | - | - | - | - |
| 20000 | 2.15 | - | - | - | - |

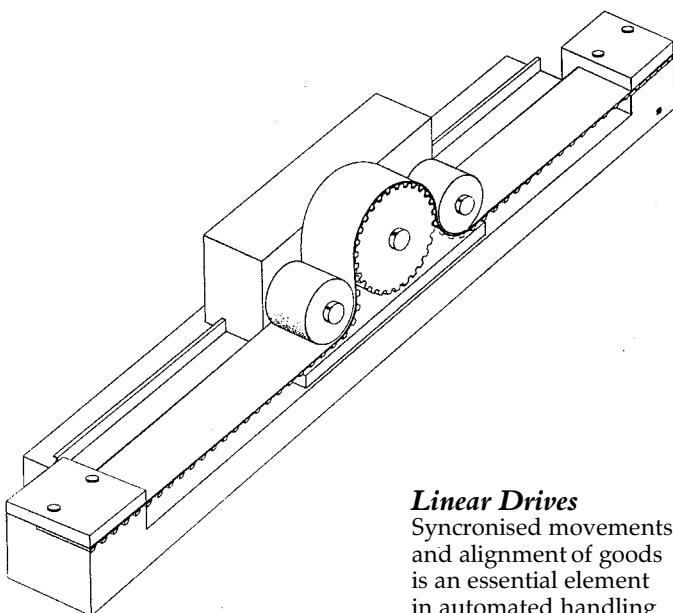
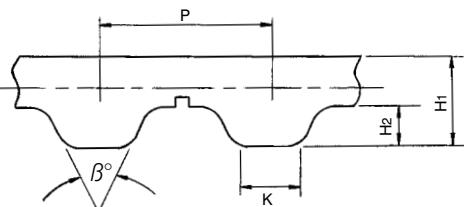
Open Ended and Special Construction Polyurethane Timing Belts



Open ended belts are manufactured in a continuous process with the steel tension members running parallel to the edges. The belts are manufactured in standard 50 metre rolls, but longer lengths are available to request. The open ended belts are normally used for reciprocating linear motions such as robotics.

The open ended belts can also be used to produce long length endless belts which are produced using an automated, precision cut vee finger joint which is weld joined. The strength of the join is by the polyurethane, but due to the large contact area loads up to 50% of the belt capacity can still be transmitted, whilst excellent flexibility and smooth running are retained. The joined belts are mainly used on Conveyor applications, and the addition of welded attachments and special backing materials extends application opportunities.

Standard Tooth Forms available



Linear Drives

Synchronised movements and alignment of goods is an essential element in automated handling systems.

| Belt Type | Pitch p | H ₁ | H ₂ | K | B° |
|-----------|---------|----------------|----------------|-------|----|
| T5 | 5.00 | 2.20 | 1.20 | 1.80 | 40 |
| T10 | 10.00 | 4.50 | 2.50 | 3.50 | 40 |
| AT5 | 5.00 | 2.70 | 1.20 | 2.50 | 50 |
| AT10 | 10.00 | 5.00 | 2.50 | 5.00 | 50 |
| AT20 | 20.00 | 8.00 | 5.00 | 10.00 | 50 |
| XL | 5.08 | 2.25 | 1.25 | 1.35 | 50 |
| L | 9.53 | 3.50 | 1.90 | 3.20 | 40 |
| H | 12.70 | 4.30 | 2.30 | 4.40 | 40 |

Standard Widths Metric Belts - width in mm

| Belt Type | 6 | 10 | 16 | 25 | 32 | 50 | 75 | 100 | 150 | 200 |
|-----------|---|----|----|----|----|----|----|-----|-----|-----|
| T5 | • | • | • | • | • | • | • | | | |
| T10 | | | • | • | • | • | • | • | | |
| AT5 | • | • | • | • | • | • | • | • | | |
| AT10 | | | • | • | • | • | • | • | • | |
| AT20 | | | | • | • | • | • | • | • | • |

Standard Widths Imperial Belts - width in 0.01"

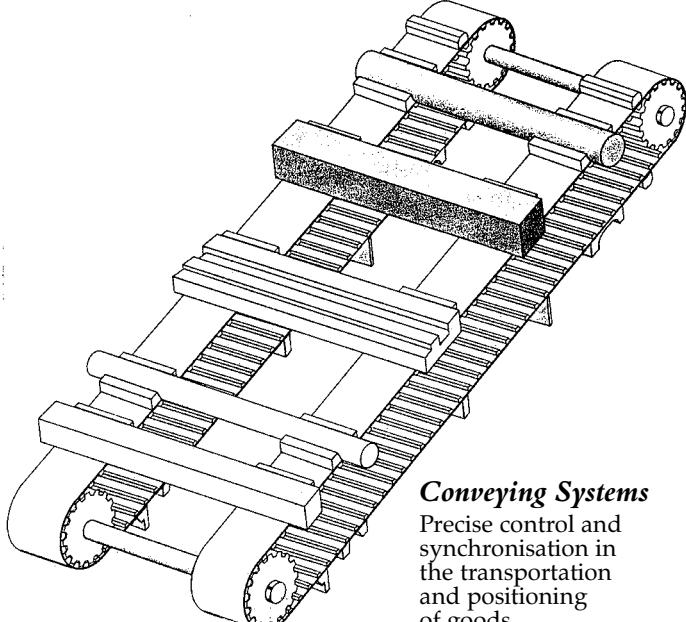
| Belt Type | 025 | 031 | 037 | 050 | 075 | 100 | 150 | 200 |
|-----------|-----|-----|-----|-----|-----|-----|-----|-----|
| XL | • | • | • | • | • | • | | |
| L | | | • | • | • | • | • | |
| H | | | | • | • | • | • | • |

Special Belt Backings

To assist in the movement of many products there is a range of specialist materials which can be bonded to the back of all sizes of Polyurethane Belts. Thickness up to 15mm enable profiling of the backing to transport special shapes, such as drawing tube from extrusion process.

Materials available with coefficient of friction varying for 0.3μ to 1.3μ and hardness 70° ShA down below 35° ShA, including open cellular materials which accommodate profile changes. Materials include polyurethane in various grades and hardness values, Linatex, Neoprene for haul-off applications, PVC, Natural Rubbers and Silicon Rubber in flat finish or honeycomb construction for higher grip.

Materials, profiles and constructions are available to cater for almost every application.



Conveying Systems

Precise control and synchronisation in the transportation and positioning of goods.

Consult Cross and Morse for the best solution to your conveying application